



Assumptions

steady state conditions; heat distributed evenly over entire cylinder; no heat lost to surroundings;
uniform heat flux

$$Q := 2500 \text{ kW}$$

Heat to be removed

Coolant is Water

$$c := 4180 \frac{\text{J}}{\text{kg} \cdot \text{K}}$$

Specific heat

$$\rho := 998 \frac{\text{kg}}{\text{m}^3}$$

Density

$$k_w := .6154 \frac{\text{W}}{\text{K} \cdot \text{m}}$$

Thermal conductivity

$$\mu := .000798 \text{ N} \cdot \frac{\text{s}}{\text{m}^2}$$

Dynamic viscosity

Flow Rate Calculation

$$T_1 := 293 \text{ K}$$

Coolant inlet temperature
(assumed)

$$T_2 := 313 \text{ K}$$

Coolant outlet temperature
(assumed)

$$\dot{m} := \frac{Q}{c \cdot (T_2 - T_1)} \quad \dot{m} = 29.904 \frac{\text{kg}}{\text{s}}$$

$$\text{flowrate} := \frac{\dot{m}}{\rho} \quad \text{flowrate} = 474.943 \text{ gpm}$$

$$\text{flowrate} = 29.964 \frac{\text{L}}{\text{s}}$$

Cooling Cylinder

$$L_1 := 10 \text{ m}$$

Water flow length (inlet
and outlet at same end)

$$r_i := 35 \text{ cm}$$

Inside radius

$$r_o := 57 \text{ cm}$$

Outside radius

$$A_C := \pi \cdot (r_o^2 - r_i^2) \quad A_C = 0.636 \text{ m}^2$$

Cylinder cross-sectional area

Tubes

$$d := .622 \text{ in}$$

Tube diameter (1/2in pipe, schedule 10)

$$v := 5 \frac{\text{m}}{\text{s}}$$

Velocity of coolant (assumed)

$$A_s := \pi \cdot d \cdot L_1$$

Surface area

$$n := \frac{\left(\frac{\text{flowrate}}{v} \right)}{\left(\pi \cdot \frac{d^2}{4} \right)} \quad n = 30.57$$

Number of tubes required

$$R_A := \frac{\left(n \cdot \pi \cdot \frac{d^2}{4} \right)}{A_c} \quad R_A = 9.425 \times 10^{-3}$$

Area ratio

.94 percent of the cross section is water

Heat Transfer Coefficient

$$Re := v \cdot d \cdot \frac{\rho}{\mu} \quad Re = 9.879 \times 10^4$$

Reynold's number

$$Pr := c \cdot \frac{\mu}{k_w}$$

Prandtl number

$$f := (.79 \cdot \ln(Re) - 1.64)^{-2}$$

Friction factor

$$Nu_d := \frac{\left[\left(\frac{f}{8} \right) \cdot (Re - 1000) \cdot Pr \right]}{\left[1 + 12.7 \cdot \left(\frac{f}{8} \right)^{.5} \cdot \left(Pr^{\frac{2}{3}} - 1 \right) \right]}$$

Nusselt number

$$h_1 := Nu_d \cdot \frac{k_w}{d} \quad h_1 = 2.062 \times 10^4 \frac{\text{W}}{\text{m}^2 \cdot \text{K}}$$

Heat transfer coefficient

$$\text{delT} := \frac{Q}{(h_1 \cdot A_s) \cdot n} \quad \text{delT} = 7.991 \text{ K}$$

Actual temperature difference from inlet to outlet



