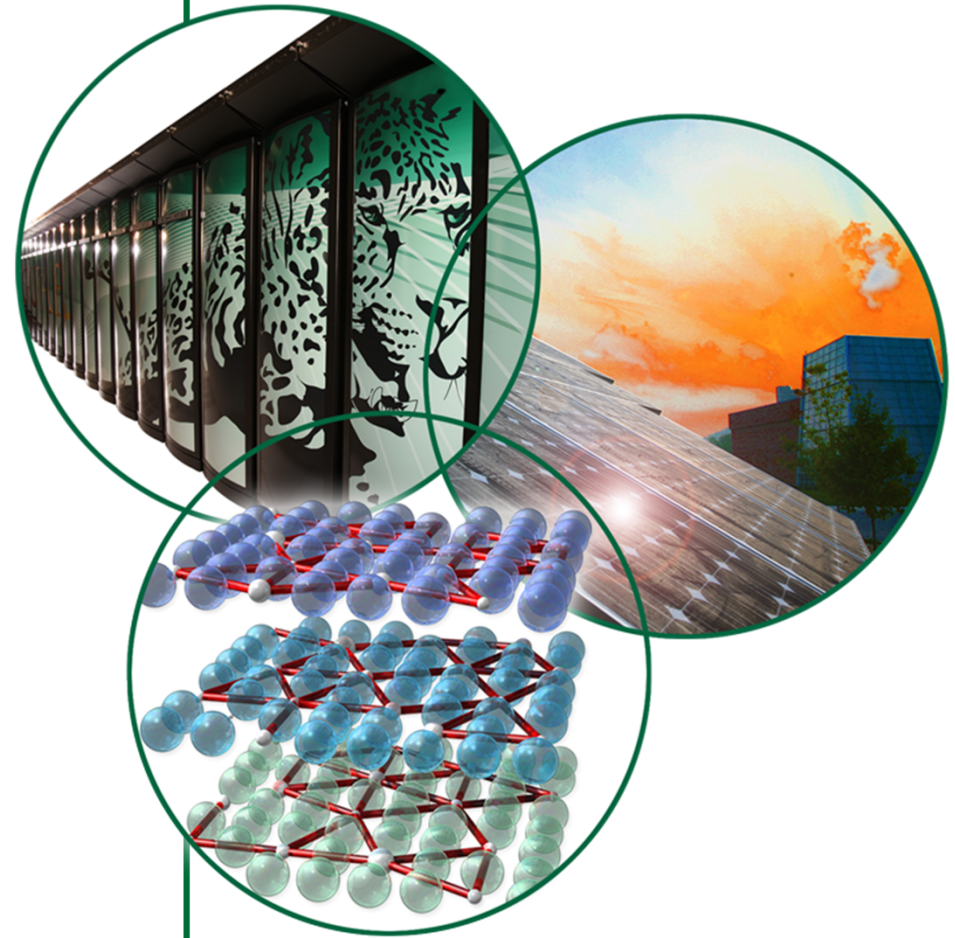


# Neutrino Factory Mercury Vessel: Initial Cooling Calculations

**V. Graves**

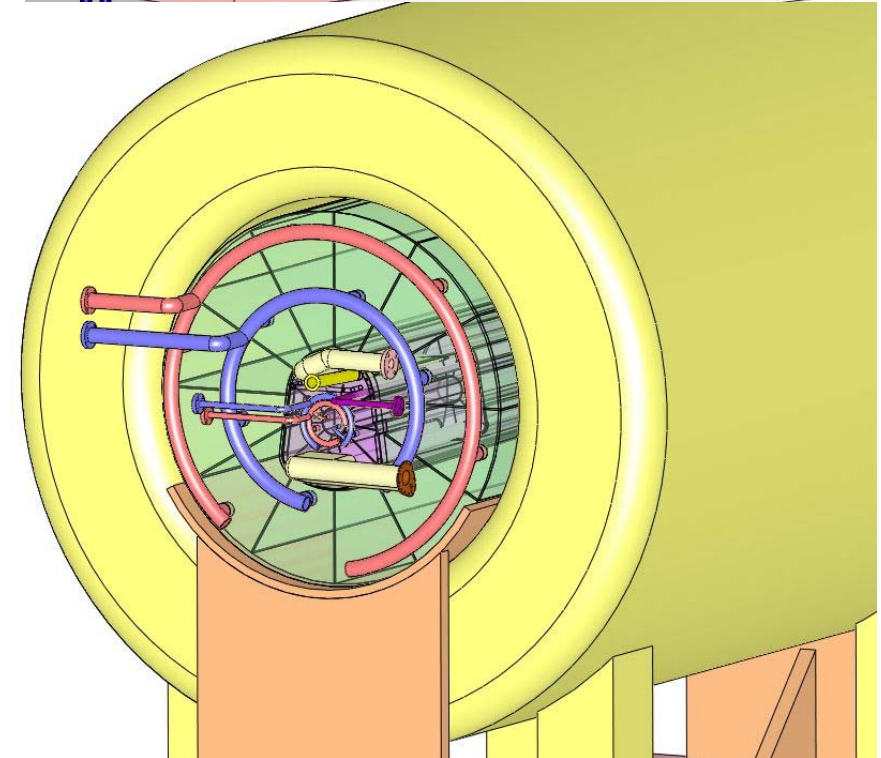
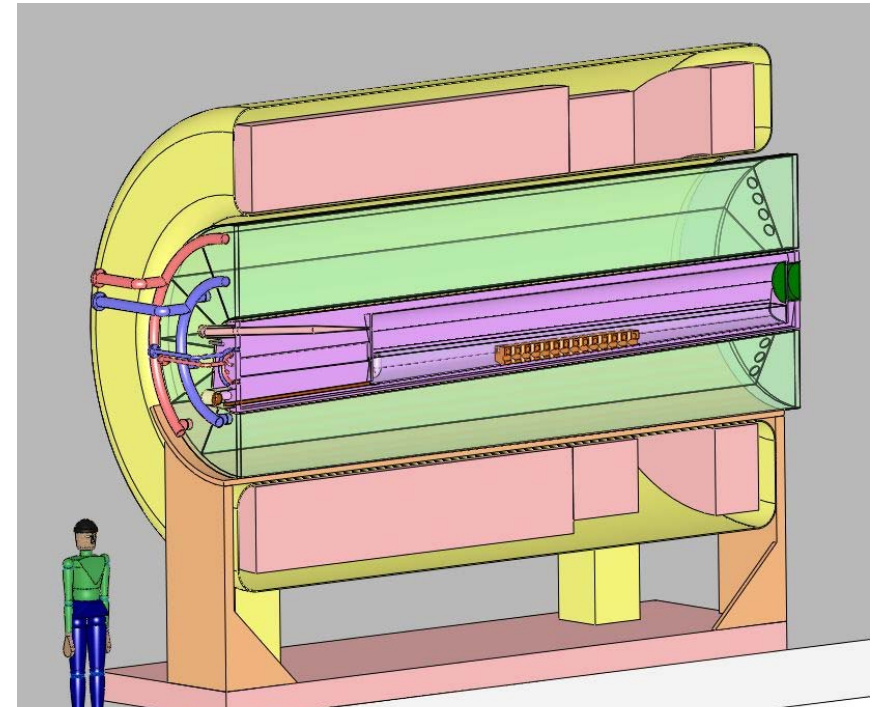
**Target Studies**

**Nov 15, 2012**



# Target System Review

- **Current mechanical concept incorporates independent mercury and shielding modules**
- **Separates functionality, provides double mercury containment, simplifies design and remote handling**
- **Each vessel assumed to be cooled with Helium**
  - **Shielding vessel filled with tungsten beads**
  - **Mercury vessel cooling chambers empty**
- **Purpose: take an initial look at the cooling issues**



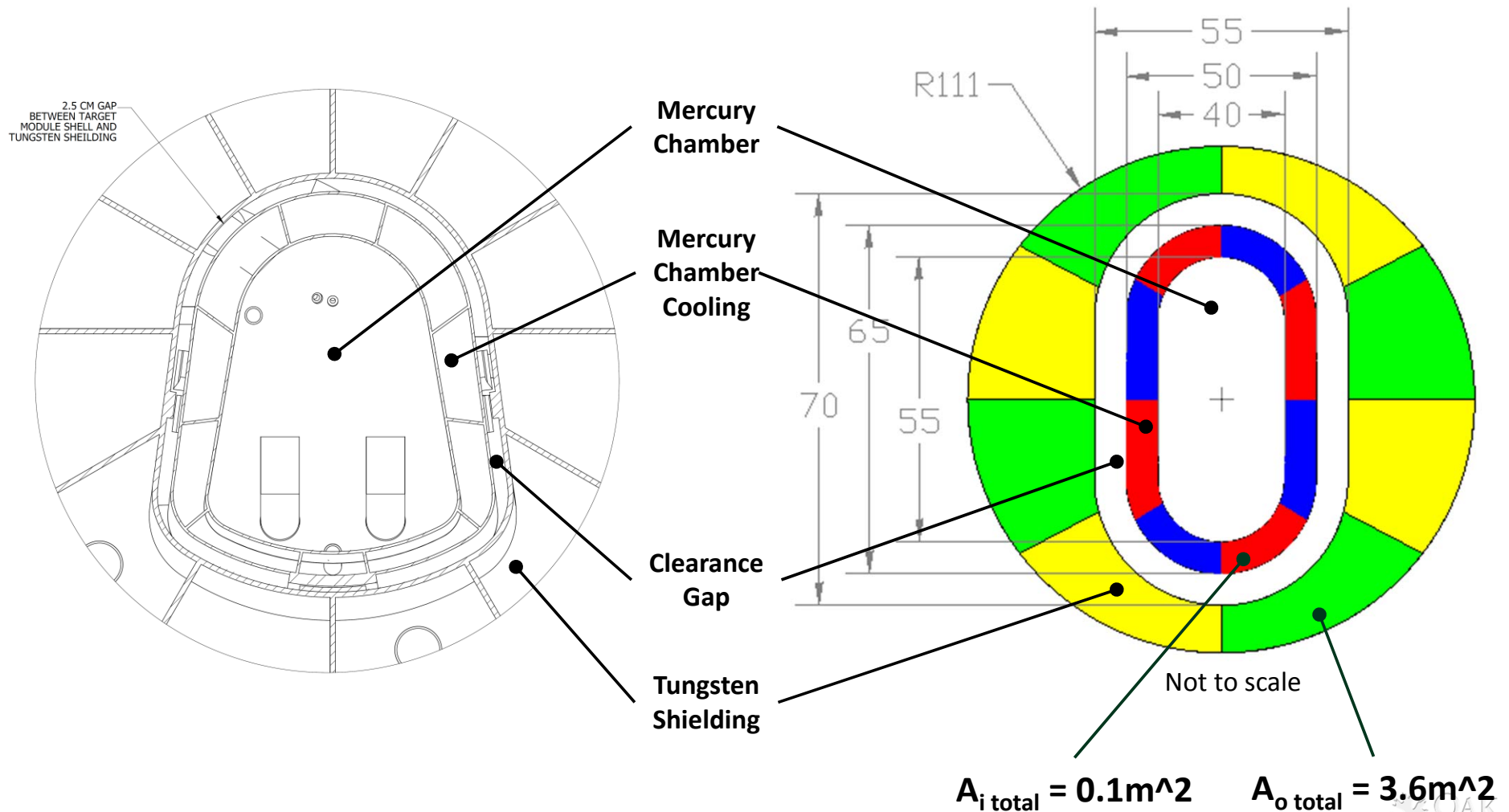
# Helium Properties @ 20C

Property	Value	Unit
Density ( $\rho$ )	0.16674	kg/m <sup>3</sup>
Dynamic Viscosity ( $\mu$ )	1.9561E-5	kg/m-s
Kinematic Viscosity ( $\nu$ )	1.1731E-4	m <sup>2</sup> /s
Specific heat (Cp)	5193	J/kg-K
Conductivity (k)	0.14786	W/m-K
Prandtl number	0.68700	
Thermal Diffusivity ( $\kappa$ )	1.7120E-4	m <sup>2</sup> /s
Thermal Expansion Coefficient ( $\alpha$ )	3.4112E-3	1/K

<http://www.mhtl.uwaterloo.ca/old/onlinetools/airprop/airprop.html>

# Analysis Model Simplification

- First-order cooling analysis based on simplified geometry model
- Break inner and outer regions into supply/return channels of equal areas within each region



# Helium Mass Flow Rates

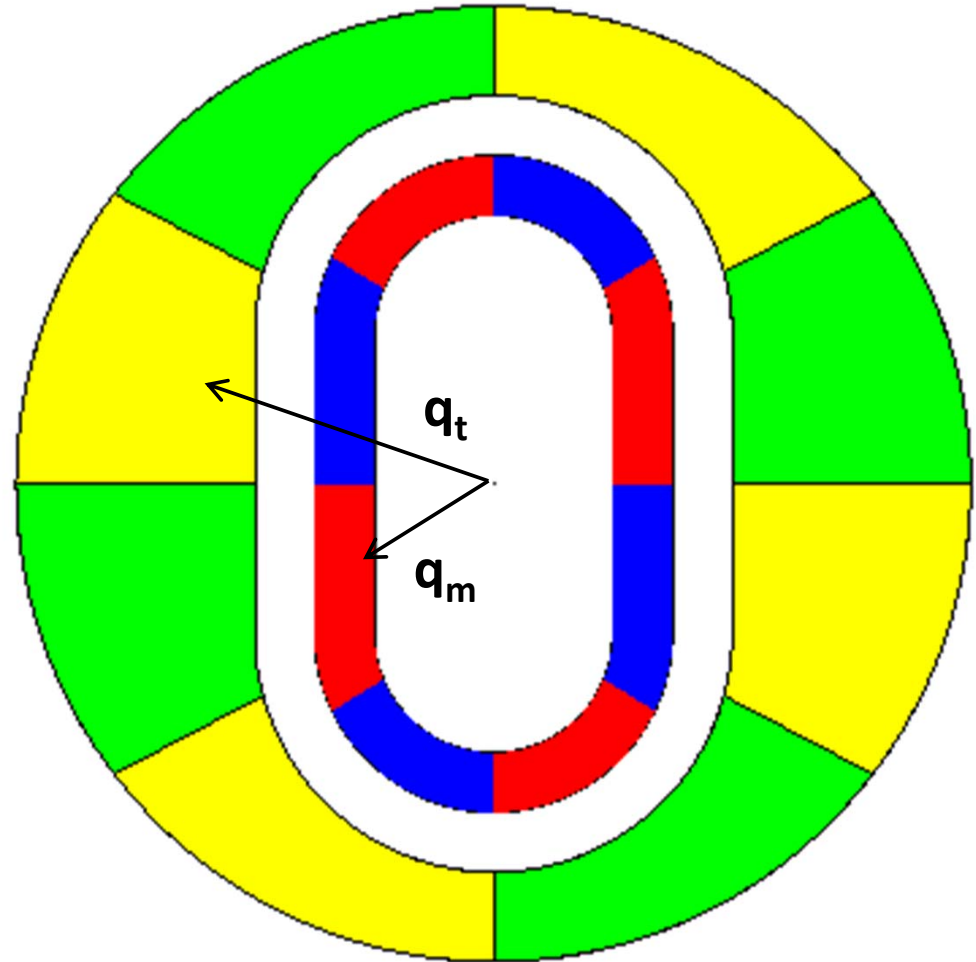
$$q = \dot{m}C_p\Delta T$$

- **Assumptions**

- $q_t = 1.5 \text{ MW}$
- $q_m = 0.5 \text{ MW}$
- $\rho = 0.16674 \text{ kg/m}^3$
- $C_p = 5193 \text{ J/kg-K}$
- Helium  $\Delta T \leq 100\text{C}$
- Helium velocity  $\leq 100 \text{ m/s}$

$$\dot{m}_t = \frac{1.5E6}{5193*100} \approx 3 \text{ kg / s}$$

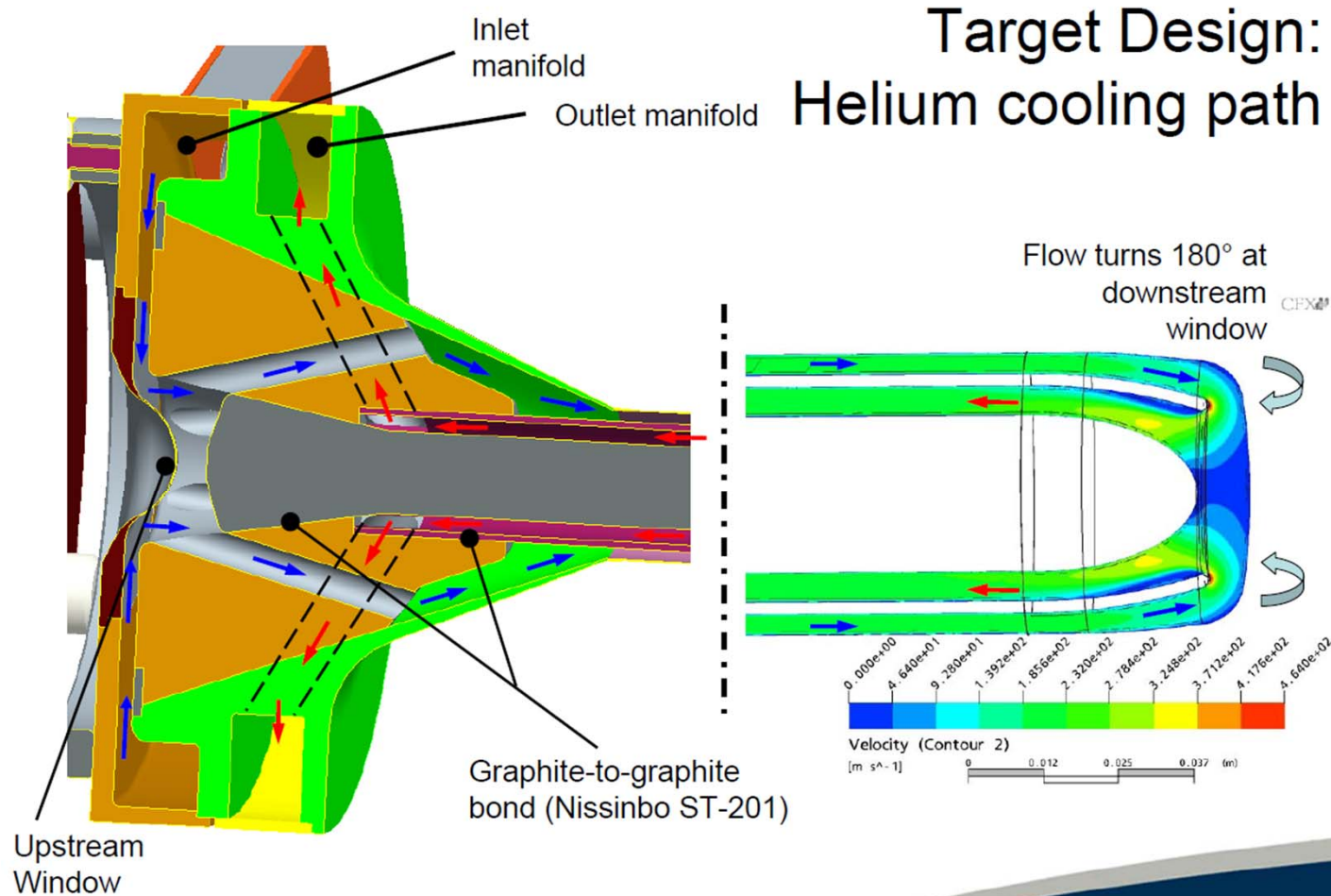
$$\dot{m}_m = \frac{0.5E6}{5193*100} \approx 1 \text{ kg / s}$$



1kgHe @ STP = 6 m<sup>3</sup>

# T2K Target Design

- Required flow rate 32 g/s
- Minimize dP (max 0.8 bar) due to high flow rate (avg = 200 m/s)

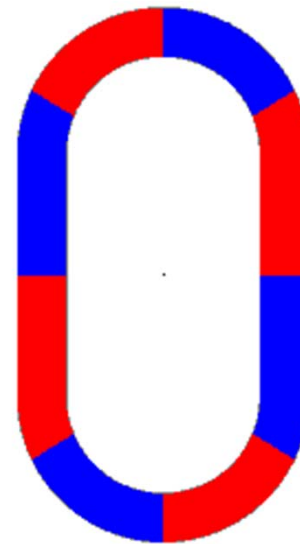


2<sup>nd</sup> Oxford-Princeton High-Power Target Workshop  
Mike Fitton

# Mercury Vessel Calculations

- Mercury cooling chamber empty (only Helium)
- Assume 4 cooling paths (8 chambers)

$$\dot{m} = \frac{\dot{m}_m}{4} = 0.25 \text{ kg / s}$$
$$A = \frac{\dot{m}}{\rho V} = \frac{0.25}{0.16674 \times 100} = 0.015 \text{ m}^2$$
$$8A = 0.12 \text{ m}^2$$
$$A_i = 0.1 \text{ m}^2$$



Helium Supply  
Channels - Blue

Helium Return  
Channels - Red

- Area may be adequate, but asymmetric heating may be problem
- Pressure drop through system needs to be calculated

# Tungsten Shielding Vessel Calculations

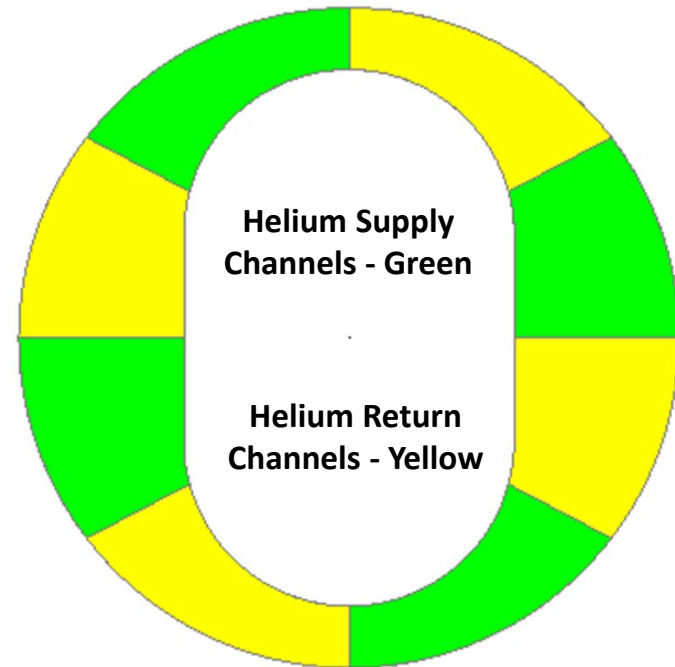
- Shielding vessel cooling chamber not empty (Tungsten spheres)
- Assume 4 cooling paths (8 chambers)

$$\dot{m} = \frac{\dot{m}_t}{4} = 0.75 \text{ kg / s}$$

$$A = \frac{\dot{m}}{\rho V} = \frac{0.75}{0.16674 \times 100} = 0.045 \text{ m}^2$$

$$8A = 0.36 \text{ m}^2$$

$$A_o = 3.6 \text{ m}^2$$



- Area adequate, may reduce helium velocity
- Pressure drop through spheres must be reviewed



# Tungsten Shielding Vessel Pressure Drop

- Ergun Equation gives pressure drop through fixed beds of uniformly sized solids

$$\frac{\Delta P}{L} = \left( \frac{\Delta P}{L} \right)_{viscous} + \left( \frac{\Delta P}{L} \right)_{kinetic}$$

$$\frac{\Delta P}{L} = 150 \frac{(1-\varepsilon)^2}{\varepsilon^3} \frac{\mu u_0}{(\phi_s d_p)^2} + 1.75 \frac{1-\varepsilon}{\varepsilon^3} \frac{\rho u_0^2}{\phi_s d_p}$$

$\Delta P$  = pressure drop

$L$  = bed length

$\mu$  = fluid viscosity

$\varepsilon$  = particle void fraction

$u_0$  = superficial fluid velocity

$\phi_s$  = particle sphericity = 1

$d_p$  = particle diameter

# Pressure Drop Results

- Assumptions

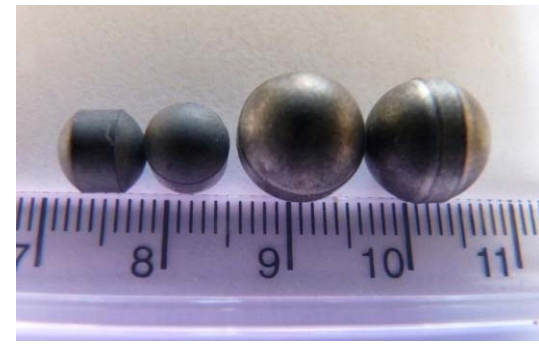
- $\epsilon = 0.4$
- $d_p = 1 \text{ cm}$

- Results indicate He pressure ~180 bar required

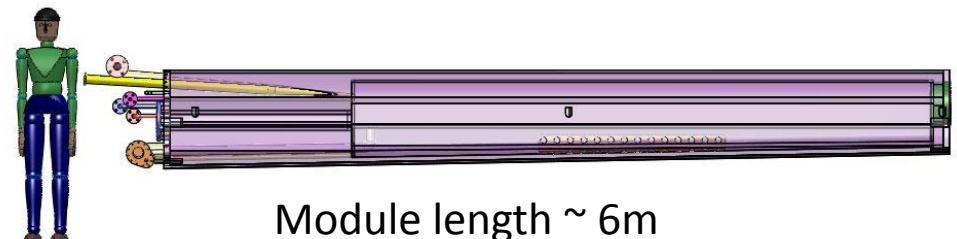
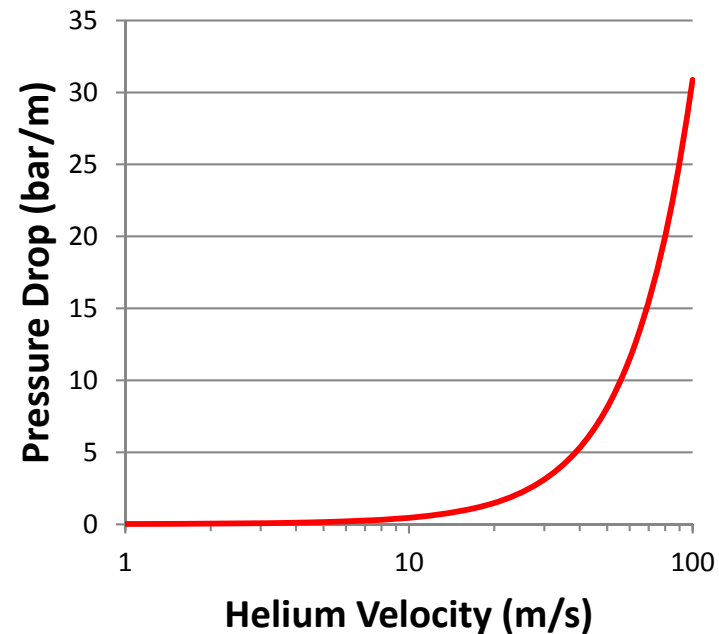
- 100m/s velocity results in large amounts of stored energy within system

- Implies we need to limit He velocity to ~ 10 m/s

- Requires 10X more flow area
- Space is available
- If need 1 s to recool the He in a heat exchanger, need 3 kg, volume = 18 m<sup>3</sup>

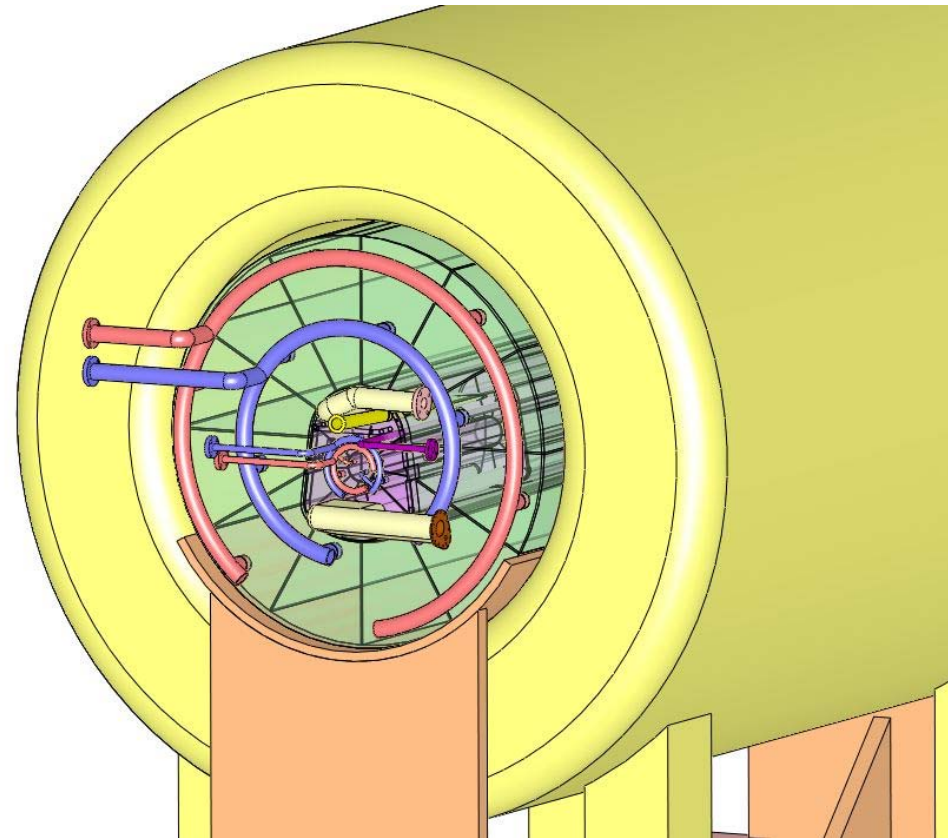


[http://www.hep.princeton.edu/~mcdonald/mumu/target/weggel/W&WC\\_spheres.pdf](http://www.hep.princeton.edu/~mcdonald/mumu/target/weggel/W&WC_spheres.pdf)



# Mechanical Complexities

- **Non-equally distributed energy deposition**
- **Complicated cooling channel geometries**
- **Flow control hardware likely to increase space requirements**
- **Implement two helium systems (one for mercury cooling, one for tungsten)?**



# Summary

- **Mercury Module now provides double-wall mercury containment with no leak path into tungsten cooling channels**
- **Helium cooling of the mercury and shielding vessels is not straightforward**
- **Initial calculations performed based on guesses for energy deposition and very simple geometry model**

